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Improving the Autonomy of a Mid-Drive Motor Electric Bicycle Based on System Efficiency Maps and Its Performance

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Abstract: Around the world, the e-bike has evolved from a recreational and sports object to an increasingly used means of transportation. Due to this, improving aspects such as range and energy efficiency has become very relevant. This article presents experimental models for the components' efficiency of a mid-drive motor e-bike (charger; battery; and controller, motor, and reduction gears subsystem), and integrates them with previously elaborated models for the chain transmission system, thus generating an overall efficiency map of the e-bike. The range of the electric bicycle is analyzed by integrating the efficiency map of the system and its performance mathematical model, aiming to determine the per unit of distance battery energy consumption. The above-mentioned calculations are applied to develop a management strategy that can determine the optimal assistance level and chain transmission ratio, maximizing range and leaving speed unaffected. The driving strategy was compared against other driving techniques using computational analysis, this allowed for the observation of the proposed strategy improving the system's range by reducing the battery energy consumption.



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Keywords: e-bike; efficiency map; range; driving strategy; performance; energy consumption

1. Introduction

Design, performance, and efficiency are core features for marketing a vehicle [1]. By considering characteristics such as generated pollution and fossil gas emissions, electric vehicles constitute clean alternatives in contrast to conventional ones [2,3]. Among them, electric bicycles stand out as a less polluting, compact, and lightweight option, with great utility for mobility in large cities worldwide [4]. However, short ranges derived from the battery's energy storage limitations bound their massification [5,6]. For this reason, the performance modeling, the study of the components' efficiency, and its integration play a fundamental role in improving range [7,8].

The basic e-bike configuration is presented by Muetze et al. [9], consisting of a controller that manages the energy flow from the battery to the electric motor, this energy works parallel to the energy produced by the cyclist. Different e-bike classification criteria are also presented in the article, namely: motor type, motor assembly, motor placement, assist type, throttle type, and battery type.

Performance is one of the most studied topics in electric bicycles. Muetze et al. [9] stated the dynamic equations of an electric bicycle and then performed physical tests with an electric bicycle equipped with sensors. As a result, they described the operation of the system presenting the maximum power, speed, and instantaneous power against variations of weight, slope, and wind speed. Evtimov et al. [10] performed experimental tests in different city routes with an electric bicycle equipped with sensors, enabling them to characterize aspects such as power, energy consumed, maximum current, maximum speed, regenerated Ah, and range. Kheirandish et al. [1] presented a PEM (polymer electrolyte membrane) fuel cell-powered electric bicycle equipped with different sensors and implemented an experimental evaluation. The results show the behavior of variables

such as system voltage, current, efficiency, and cell power. Kang et al. [11] proposed an assessment method for the engine and battery behavior of an electric vehicle using a dynamometer.

The dynamic features and power required by an electric bicycle have been studied under the effect of parameters such as rider mass, bicycle mass, air speed, crank length, wheel radius, and slope. Dynamic equations that govern an electric bicycle are used for this purpose, speed, distance traveled, and power behavior are simulated, and experimental tests are conducted to validate mathematical models [12–14]. Hung et al. [12] focus their study on a hub motor bicycle with a fixed gear ratio for the chain. Hung et al. [13] study a hub motor electric bicycle and a semi-automatic transmission, analyzing and presenting results for each possible transmission ratio. In the study, they establish a motor efficiency of 70%. Hung et al. [14] study a hub motor electric bicycle considering a variable transmission ratio for the sprocket and dynamic models for the battery.

Arango et al. [15] compare the performance of hub and mid-drive motors on electric bicycles for mountain routes. They experimentally characterize the two types of motors on a test bench, obtaining their torque, RPM, and efficiency curves for each assist level. Subsequently, they use this information and the system's dynamic equations to simulate bicycles' performance on mountain routes. For the e-bike's case, it was concluded that the motor placement is the most relevant factor for the system's energy consumption, and it was shown that, for steep roads, middle drive motors use about 18% less energy than hub motors.

Electric bicycles range and its optimization are still underdeveloped. Gebhard et al. [16] worked on improving the prediction of electric bicycle range by assessing two prediction methods considering the cyclist behavior as well as the route. Ferreira et al. [5] present a mobile application aimed to provide useful information to the cyclist, such as range prediction based on a specified route, cyclist effort management, battery charging process management, among others. De La Iglesia et al. [17] developed an intelligent engine management system to optimize the selection of assistance, intended for reducing battery power consumption. This is achieved by computing information provided by sensors on the bicycle, historical data from other cyclists, and neural networks. Its results achieved a decrease in electricity consumption along the route, saving 10.32% of consumption by controlling the electric bicycle assist level.

As showcased, multiple studies on e-bikes have mainly analyzed the external forces affecting the system (weight, rolling, and drag); however, the behavior of internal energy losses in these systems due to their components has not been analyzed yet. At the same time, no studies have been presented that optimize the operational range of the bicycle based on the knowledge of its performance, nor its components' behavior. Minav et al. [18] and Wu et al. [19] quantified the efficiency of machine components such as forklifts or electric cars using maps, curves, and punctual values. This information was used to determine the efficiency of such systems. Moreover, it was possible to analyze energy losses and enhancement opportunities in the systems. In addition, knowledge on energy accumulators and powertrain elements efficiency in electric vehicles enabled the implementation of energy-efficient driving strategies [20]. Based on this, the article presents an experimental analysis of internal energy losses for a mid-drive motor electric bicycle by characterizing its components and the system through maps, curves, and values. The integration of such information and the system's performance analysis results in the presentation of a novelty management strategy aimed at improving the electric bicycle's range by controlling the transmission ratio and assistance level.

The remaining contents of the article are ordered as follows: Section 2 presents the materials and methods used to develop and assess the proposed driving strategy; Section 3 shows the results obtained from experimental and computational studies; Section 4 showcases the discussion.

2. Materials and Methods

This section initially presents the method by which the energy flow through the bicycle is analyzed, as well as the losses in its components; then, the equations for the system's dynamic behavior and performance are presented; after that, the proposed driving strategy to increase the e-bike's range is described; and finally, a case study is showcased in which the performance of the driving strategy is compared with other driving techniques.

2.1. E-Bike Losses Characterization

To identify the energy flows through an e-bike, its transformations, and losses, different studies present their block diagram model [9,15].

Figure 1 shows the block diagram for a mid-drive motor e-bike. It presents the energy and information flows for the system regarding the energy delivered by the cyclist and the information transmitted by the cyclist.

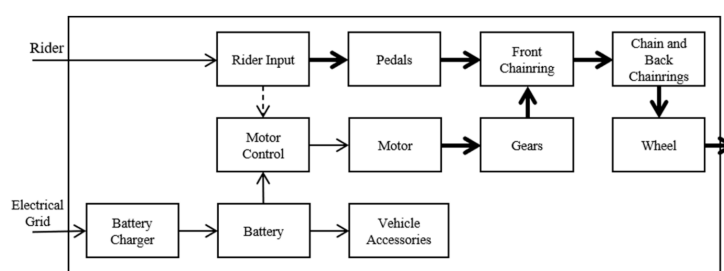


Figure 1. Block diagram for a mid-drive motor e-bike.

For this study, a commonly used e-bike and its components were subject to different experiments to characterize efficiency. Table 1 presents specifications analyzed for this e-bike.

Table 1. E-bike specifications.

Specifications		
Battery Type	Lithium-Ion	
Nominal Battery Voltage	48 V	
Battery Electric Charge	11.4 Ah	
Motor Placement	Mid-motor	
Motor Type	BLDC	
Reduction Gear Ratio	1:21.9	
Chain Wheel (number of teeth)	34	
Cassette (number of teeth)	40/35/31/27/24/21/19/17/15/13/11	
Controller Assistance Levels	9	
Wheel Diameter	27.5 in	

The e-bike was analyzed by subsystems, namely: the charger; the battery; the controller, motor, and reduction gears (CMRGs); and the chain drive. The CMRGs group was a subsystem analyzed together because their physical construction did not allow partitioning without partially or entirely affecting functionality.

2.1.1. Charger

This element draws electricity from the power grid and rectifies it to supply the battery using an AC-DC converter followed by a DC-DC converter. The charging process comprises two stages: Constant current (CC) charge and constant voltage (CV) charge. Charger losses are mainly due to conduction losses, MOSFET (metal–oxide–semiconductor field-effect transistor) and diode switching, and core losses [21,22].

The efficiency of the charger was characterized through an experimental test in which a full battery charge cycle (from 0% to 100% state of charge (SoC)) was performed. During this

process, current and voltage sensors were connected to the charger input and output (power cables and battery connector). A microcontroller board stored the sensors' measurements every minute, with each measurement corresponding to the average of 100 sensor readings. The efficiency of the charger (η_{char}) is calculated as the input energy (E_{charge} (Wh)) (taken from the power grid) and the output energy (E_{grid} (Wh)) (delivered to the battery) ratio, as shown in Equation (1). Voltage (V (V)), current (I (A)), and time (t (s)) measurements, and power (P (W)) and electrical energy E (J)) equations were used to calculate such energy measurements, this is represented with Equations (2) and (3).

$$\eta_{char} = \frac{E_{charge}}{E_{grid}} \quad (1)$$

$$P = V \times I \quad (2)$$

$$E = P \times t \quad (3)$$

2.1.2. Battery

Battery efficiency is defined as the ampere-hours removed from the battery to the ampere-hours restored to the battery ratio, for the same final and initial conditions (SoC and temperature) [23]. Factors such as parasitic reactions in its electrochemistry, battery aging, ultra-fast charges, and high working loads affect the energy delivering capacity of the battery [24]. E-bikes use different battery types such as lead, nickel-cadmium, NiMH, lithium-ion polymer, or lithium-ion, the latter ones being the most widely used [25]. Farhad et al. [26] generated efficiency maps for different lithium-ion battery families, using their results, and after determining the energy density of a battery, it is possible to establish the operating efficiency relying on the C ratio alone.

The battery's energy performance was characterized through an experimental test aimed to generate a function representing the battery efficiency as a function of the discharge current. For this purpose, the energy delivered by the battery operating at different C ratios was quantified and compared with the energy stored in the battery during the charging stage. This experiment was divided into two processes: charging and discharging. For the charging process, the electrical energy entering the battery was quantified using the calculations and methods developed for the characterization of the charger efficiency. For the discharge process, the electrical energy delivered by the battery was quantified by implementing a circuit in which the battery was connected to a variable resistive load and complementing the circuit with voltage and current sensors. The discharge rates of the tests are presented in Table 2.

Table 2. Battery discharge rates.

C Ratio	1/8 C	1/4 C	3/8 C	1/2 C	5/8 C	3/4 C	7/8 C	1 C
Battery Current (A)	1.48	2.95	4.43	5.90	7.38	8.85	10.33	11.80

Equations (2) and (3) allowed the calculation of the charge (E_{charge} (Ah)) and discharge ($E_{discharge}$ (Ah)) energy of the battery, so its efficiency at each operating point was calculated as presented in Equation (4), where η_{bat} is the battery efficiency.

$$\eta_{bat} = \frac{E_{discharge}}{E_{charge}} \quad (4)$$

2.1.3. Controller, Motor, and Reduction Gears (CMRGs)

These elements are closely linked, and their selection depends mainly on the motor. Despite the variety of existing electric motors, e-bikes are generally equipped with DC motors, with or without brushes. Brushless DC motors (BLDCs) are more advantageous

for electric bicycles than brushed DC motors, because they are more efficient and smaller, yet their control is more complex [9].

Hung et al. [25] referenced various types of controls for BLDC motors on e-bikes such as PID control, fuzzy logic control, torque control, traditional power assistance approach, and sensorless power assist control method. Regardless of the control method, the controller has always an embedded inverter, which generally uses 6 MOSFET transistors to convert electricity from DC to AC. This process involves energy heat losses due to the internal resistance of MOSFET transistors and the switching process [27,28].

Motor efficiency is represented through efficiency maps as a function of its torque and RPM [29]. This component can be coupled to gearboxes or directly driven; however, reduction gears are commonly used to decrease the motor rotation speeds respecting the bicycle wheels, thereby generating a more efficient operation. Spur gears are common in mid-drives and hub-drives [30,31]. The efficiency of spur gears is over 98.0% without any noticeable variation as the load or speed changes [32,33].

This subsystem was analyzed through an experiment aimed at developing the efficiency map of the CMRGs group. For this purpose, the electrical power drawn by the controller from the battery was compared with the mechanical power delivered by the gear motor at the output shaft. To conduct the experiment, a voltmeter and an amperemeter were connected to the controller input. A torque meter was also coupled directly to the gear motor output shaft, and this to a disc brake. The mechanical assembly of the experiment is shown in Figure 2.



Figure 2. Controller, motor, and reduction gears (CMRGs) experimental set-up.

The experiment went through the entire map of torque vs. RPM possibilities by varying the braking torque, with values from 0 to 70 Nm, and varying the motor speed through the nine assist levels that the controller possesses. During the tests, the electrical power (P_{elec} (W)) was calculated using voltage and current measurements, while mechanical power (P_{mec} (W)) was calculated with data provided by the torque meter (torque and RPM). With this, the efficiency of the CMRGs group (η_{cmrgs}) was quantified for each operation point using Equation (5).

$$\eta_{cmrgs} = \frac{P_{mec}}{P_{elec}} = \frac{T \times \omega}{V \times I} = \frac{T \times RPM \times 2\pi}{V \times I \times 60} \quad (5)$$

where T (Nm) is the torque on the CMRGs output shaft and ω (rad/s) is the output shaft angular speed. To facilitate the use of the information provided by the CMRGs efficiency map, data obtained in the experiment was subject to a linear regression process which sought to represent it through parametric equations as a function of the output torque and RPM.

2.1.4. Chain Drive

The derailleur system is the most used chain drive configuration for bicycles, presenting efficiencies ranging between 80% and 98%. Chain drive efficiency was represented by different functions reported by Spicer [34]. His results demonstrated that the efficiency of a chain drive varies respecting the reciprocal of the tension to which it is exposed, where the

greater the tension the greater the efficiency. Factors such as RPM, power, torque, chain-rings teeth count, and chain misalignment have minor incidences over the performance of the mechanism. This information was used to simulate the chain drive efficiency for different transmission ratios in this study case [34–36].

2.1.5. Global System Efficiency

The e-bike system's global efficiency (η_{eb}) is presented as a map, and it is calculated based on the integration of the efficiency of each subsystem, namely: charger, battery, CMGRs, and chain drive. The efficiency of the system at each operating point was mathematically represented as:

$$\eta_{eb} = \eta_{char} \times \eta_{bat} \times \eta_{cmrgs} \times \eta_{chain} \quad (6)$$

2.2. Bicycle Dynamics

The mathematical model used to evaluate the dynamics, and the analyzed system simulation are described below. First, the external forces and the equation governing the motion of the system are presented; then the equations characterizing its velocity and displacement are presented; and finally, the equations representing the performance of its components and their energy consumption are presented.

The dynamic diagram presenting the forces that govern the movement of an electric bicycle is shown in Figure 3, and the equation which represents this behavior is described as:

$$F_p - (F_r + F_s + F_w) = m \frac{d^2x}{dt^2} \quad (7)$$

$$(F_p) - \left(m g C_r \cos(\alpha) + m g \sin(\alpha) + \frac{1}{2} C_d \rho A v_{ga}^2 \right) = m \frac{d^2x}{dt^2} \quad (8)$$

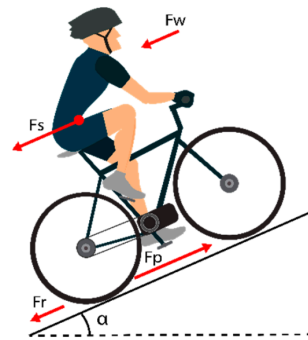


Figure 3. Dynamic diagram of an electric bicycle.

Where F_p (N) is the propulsion force, F_r (N) is the rolling resistance force, F_s (N) is the force generated by gravity, F_w (N) is the aerodynamic drag force, m (kg) is the system mass, x (m) is the distance, t (s) is the time, g (m/s^2) is the gravitational acceleration, C_r is the rolling coefficient, α ($^\circ$) is the road slope angle, C_d is the drag coefficient, ρ (kg/m^3) is the air density, A (m^2) is the system frontal area, and v_{ga} (m/s) is the relative speed of the system respecting the air.

By knowing the terms in Equation (7), and integrating them, the system speed v ($\frac{m}{s}$) is obtained through algebraic differences as:

$$\frac{(F_p - (F_r + F_s + F_w)) \times dt}{m} = \Delta v \quad (9)$$

$$v = v_{ini} + \Delta v \quad (10)$$

Where $\Delta v \left(\frac{m}{s}\right)$ is the speed differential and $v_{ini} \left(\frac{m}{s}\right)$ is the previous system speed. The integration of the system speed for a delta time allows the calculation of the distance traveled during that instant $\Delta x \left(m\right)$, and this enables the calculation of the accumulated distance.

$$\Delta x = v_{end} \times dt \quad (11)$$

$$x = x_{ini} + \Delta x \quad (12)$$

Once the equations governing the dynamics of the system, its velocity, and displacement have been presented, the relationships representing the performance of its components and their energy consumption are presented.

The torque received by the wheel $T_w \left(Nm\right)$ is calculated with the propulsive force and the wheel radius $r_w \left(m\right)$ as:

$$T_w = F_p \times r_w \quad (13)$$

Knowing the torque received by the wheel allows the chain tension $F_{t_{chain}} \left(Nm\right)$ and its efficiency η_{chain} to be determined by using the functions presented by Spicer [34], the radius of the sprocket in the cassette $r_{s_i} \left(m\right)$ and the following relationships:

$$F_{t_{chain}} = T_w / r_{s_i} \quad (14)$$

$$\eta_{chain} = f(F_{t_{chain}}) \quad (15)$$

The efficiency of the chain, its transmission ratio TR_{chain} , and the torque received by the wheel are used to determine the torque on the crank spindle $T_{crank_{spindle}} \left(Nm\right)$, calculated as:

$$T_{crank_{spindle}} = T_w \times \eta_{chain} \times TR_{chain} \quad (16)$$

The torque on the crank spindle is the sum of the torque delivered by the cyclist $T_{cyclist} \left(Nm\right)$ and the torque delivered by the controller, motor, reduction gears (CMRGs) group $T_{CMRGs} \left(Nm\right)$:

$$T_{crank_{spindle}} = T_{cyclist} + T_{CMRGs} \quad (17)$$

The torque delivered by the cyclist is presented as a relationship between its cadence $RPM_{cyclist}$ and its capacity to deliver power $P_{cyclist} \left(W\right)$, expressed as:

$$T_{cyclist} = f(RPM_{cyclist}, P_{cyclist}) \quad (18)$$

The RPM on the wheel RPM_w are dependent on the system speed and the wheel radius, and are calculated as:

$$RPM_w = v * \frac{60}{2\pi * r_w} \quad (19)$$

The RPM on the crank spindle $RPM_{crank_{spindle}}$ are dependent on the chain transmission ratio and the wheel RPM, so they are calculated as:

$$RPM_{crank_{spindle}} = \frac{RPM_w}{TR_{chain}} \quad (20)$$

Using the CMRGs efficiency map, the RPM at the crank spindle, and the torque at the CMRGs, its efficiency η_{CMRGs} is determined as:

$$\eta_{CMRGs} = f(T_{CMRGs}, RPM_{crank_{spindle}}) \quad (21)$$

Through the relations presented in Equation (5), the mechanical power of the CMRGs $P_{mec_{CMRGs}} \left(w\right)$ is calculated employing its torque and RPM. The electrical power of the CMRGs $P_{elec_{CMRGs}} \left(w\right)$ is also determined by associating its mechanical power and its efficiency. In turn, the current received by the CMRGs $I_{CMRGs} \left(A\right)$ is calculated using the electrical power of the CMRGs and the battery voltage $V_{bat} \left(V\right)$.

The energy charged to the battery is calculated with the electrical energy delivered to the CMRGs and the efficiency of the battery. The same relationship applies for its power and current, so the current removed from the battery I_{bat} (A) is calculated as:

$$I_{bat} = I_{CMRGs} / \eta_{bat} \quad (22)$$

The voltage product integration and the current removed from the battery for a differential of time indicates the watt-hours consumed Wh , this is expressed as:

$$Wh = \sum V_{bat} \times I_{bat} \times \Delta t \quad (23)$$

Furthermore, by relating the watt-hours consumed with the distance differential for the same time delta, it is possible to identify the ratio of watt-hours consumed per unit of distance.

$$rel = \Delta Wh / \Delta x \quad (24)$$

2.3. Bicycle Performance

To fulfill the necessary relationships to evaluate the behavior of an electric bicycle, the performance equations are described below. The power developed both by the electric motor and the cyclist is used to overcome the power losses on the components (P_{losses}), as well as that generated by the slope (P_s), rolling (P_r), and drag (P_d) [15]. This relationship is expressed as:

$$P_{total} = P_{cyclist} + P_{em} = P_{losses} + P_r + P_s + P_d \quad (25)$$

where P_{total} (W) is the sum of the power produced by the cyclist $P_{cyclist}$ (W) and the power produced by the electric motor P_{em} (W). The power consumed to overcome slope, rolling, and drag is calculated as:

$$P_i = F_i \times v \quad (26)$$

where P_i (W) stands for each power, F_i (N) is each force, and v ($\frac{m}{s}$) is the relative speed to the ground. The power losses for each component are expressed as:

$$P_{losses} = P_{in} - P_{out} \quad (27)$$

$$P_{out} = P_{in} \times \eta \quad (28)$$

where P_{in} (W) is the power that enters each component, P_{out} (W) is the power delivered by the component, and η is the component efficiency.

2.4. Driving Strategy

A driving strategy is proposed to enhance the range of the system, modeled as an optimization problem intended to determine the operational point of the system where the best ratio of per unit of distance covered energy consumed is given. The strategy is based on indicating the assistance level and the chain transmission ratio depending on the slope to which the system is submitted. For this purpose, the performance, and dynamic equations (equations to which the problem is subject) are used. As domain restrictions, there are:

$$v_{min} \leq v \leq v_{max} \quad (29)$$

$$RPM_{crank_{spindle_{min}}} \leq RPM_{crank_{spindle}} \leq RPM_{crank_{spindle_{max}}} \quad (30)$$

$$TR_{chain} \in \left\{ \frac{34}{40} : \frac{34}{35} ; \frac{34}{31} ; \frac{34}{27} ; \frac{34}{24} ; \frac{34}{21} ; \frac{34}{19} ; \frac{34}{17} ; \frac{34}{15} ; \frac{34}{13} ; \frac{34}{11} \right\} \quad (31)$$

$$AL \in \{1; 2; 3; 4; 5; 6; 7; 8; 9\} \quad (32)$$

$$0 \leq T_{CMRGs} \leq T_{CMRGs_{max}} \quad (33)$$

$$T_{cyclist} \leq T_{CMRGs} \quad (34)$$

Bicycle speed between the desired speed ranges must be assured; the RPM on the crank spindle must be in range with the maximum and minimum angular speeds generated by the assistance levels (AL); the chain transmission ratio will only have values between the possible chain wheel–cassette combinations; the assistance levels are determined by the bicycle controller; the torque at the CMRGs cannot take values higher than its maximum capacity; and the torque delivered by the CMRGs must be greater or equal to the torque delivered by the cyclist, thus enabling the electrical system to execute most of the work.

2.4.1. Maximization Function

The electrical power and its transformation through the system is controlled by managing the chain transmission ratio and the assistance level (AL). The analysis of possible combinations for these two elements allows determining the possible operational points where the system reaches static equilibrium. Therefore, it is possible to identify the combination offering the best per unit of distance energy consumed ratio to maximize its range. Consequently, the function to be maximized is expressed as:

$$\max_{RDT, AL} \text{range} (TR_{chain}, AL) \quad (35)$$

2.4.2. Chain Transmission Ratio and Assistance Level Selection Algorithm

The following algorithm is used to identify the optimum assistance level and chain transmission ratio:

1. Identify the slope.
2. Generate a row vector of possible system desired speeds (v_j), for $v_{min}:\Delta v:v_{max}$ (m/s).

$$v_j = [v_{min} \quad \dots \quad v_{max}] \quad (36)$$

3. Calculate a distance traveled (d_j) row vector by integrating each position of v_j vector by a differential of time.
4. Calculate an RPM on the wheel (RPM_{w_j}) row vector by using Equation (19) at each position of the vector v_j .
5. Generate an RPM on the crank spindle ($RPM_{crankspindle_{i,j}}$) array in which each row represents the RPM_{w_j} vector related to each gear ratio, through Equation (20).

$$RPM_{crankspindle_{i,j}} = \begin{bmatrix} \frac{RPM_{w_1}}{TR_{chain_1}} & \dots & \frac{RPM_{w_n}}{TR_{chain_1}} \\ \vdots & \ddots & \vdots \\ \frac{RPM_{w_1}}{TR_{chain_m}} & \dots & \frac{RPM_{w_n}}{TR_{chain_m}} \end{bmatrix} \quad (37)$$

$$= \begin{bmatrix} RPM_{crankspindle_{1,1}} & \dots & RPM_{crankspindle_{1,n}} \\ \vdots & \ddots & \vdots \\ RPM_{crankspindle_{m,1}} & \dots & RPM_{crankspindle_{m,n}} \end{bmatrix}$$

6. Generate a propulsion force (F_{p_j}) row vector on the bicycle based on Equation (8), system parameters and considering a static equilibrium (no acceleration) system applied to each position of the vector v_j .
7. Calculate a torque on the wheel (T_{w_j}) row vector with each position of the vector F_{p_j} , Equation (13), and the wheel radius.

8. Generate a torque on the crank spindle ($T_{crank_spindle_{i,j}}$) array, in which each row represents the vector T_{w_j} related to each chain transmission ratio and the chain efficiency by using Equations (14)–(16).

$$T_{crank_spindle_{i,j}} = \begin{bmatrix} T_{crank_spindle_{1,1}} & \cdots & T_{crank_spindle_{1,n}} \\ \vdots & \ddots & \vdots \\ T_{crank_spindle_{m,1}} & \cdots & T_{crank_spindle_{m,n}} \end{bmatrix} \quad (38)$$

9. Calculate a cyclist torque ($T_{cyclist_{i,j}}$) array with each position of array $RPM_{crank_spindle_{i,j}}$, cyclist power, and Equation (18).
10. Calculate a torque delivered by CMRGs ($T_{CMRGs_{i,j}}$) array by applying Equation (17) for each position of arrays $T_{crank_spindle_{i,j}}$ and $T_{cyclist_{i,j}}$.
11. Calculate a mechanical power of the CMRGs group ($P_{mecCMRGs_{i,j}}$) array by using Equation (5) for each position of arrays $T_{CMRGs_{i,j}}$ and $RPM_{crank_spindle_{i,j}}$.
12. Calculate an efficiency in the CMRGs group ($\eta_{CMRGs_{i,j}}$) array by applying, for each position of arrays $T_{CMRGs_{i,j}}$ and $RPM_{crank_spindle_{i,j}}$, the function expressed in Equation (21).
13. Calculate a Watt-hours removed from the battery ($Wh_{i,j}$) array using, for each position of arrays $P_{mecCMRGs_{i,j}}$ and $\eta_{CMRGs_{i,j}}$, Equations (5), (22), and (23).
14. Calculate an energy consumption per unit of distance ($rel_{Wh/d_{i,j}}$) array by dividing each row of array $Wh_{i,j}$ by vector d_j .

$$rel_{\frac{Wh}{d_{i,j}}} = \begin{bmatrix} Wh_{1,1} \div d_1 & \cdots & Wh_{1,n} \div d_n \\ \vdots & \ddots & \vdots \\ Wh_{m,1} \div d_1 & \cdots & Wh_{m,n} \div d_n \end{bmatrix} = \begin{bmatrix} rel_{\frac{Wh}{d_{1,1}}} & \cdots & rel_{\frac{Wh}{d_{1,n}}} \\ \vdots & \ddots & \vdots \\ rel_{\frac{Wh}{d_{m,1}}} & \cdots & rel_{\frac{Wh}{d_{m,n}}} \end{bmatrix} \quad (39)$$

15. Identify the minimum value in array $rel_{Wh/d_{i,j}}$ and its position (i, j). The ideal chain transmission ratio in which the system should operate is identified with the (i) position. The ideal crank spindle RPM is identified by finding the (i, j) position in array $RPM_{crank_spindle_{i,j}}$. The ideal value is selected using this value and the curves representing each assistance level.

2.5. Study Case

Multiple simulations were performed when implementing the dynamics of the system to contrast the operation of the driving strategy proposed with other driving techniques. The simulations employed the characteristics of the electric bicycle described above. The remaining parameters of the system are presented in Table 3. Cyclist power was selected so that no fatigue was generated.

Table 3. System parameters.

System Mass (kg)	C_r	C_d	ρ (kg/m ³)	A (m ²)	v_{wind} (m/s)	Cyclist Power (W)
100	0.0055	1.1	1.19	0.51	0	100

Three management strategies were studied. The first one corresponded to the one proposed in this article, in which the assistance level and the transmission ratio were determined based on the slope and the stated algorithm. The second one sought to keep a cadence between 50 and 60 RPM by varying the chain transmission ratio [37]. The third one sought to keep a cadence between 70 and 90 RPM by varying the chain transmission ratio [38]. The second and third strategies correspond to riding habits commonly used by

untrained cyclists, and, in both cases, the assistance level was kept constant and selected aiming to generate a cadence like the desired for each strategy.

To assess the behavior of the different driving strategies, eight 10 km long and constant slope routes were used. The slope was the only factor that changed among routes, ranging from 0% to 12.28%.

The performances of strategy two (cadence between 50 and 60) and strategy three (cadence between 70 and 90) were initially characterized in the eight mentioned routes, initially using e-bike power alone and then using e-bike and cyclist power, regarding electric energy consumed, energy delivered by the cyclist, and average speed. Afterwards, to contrast the proposed driving strategy respecting the other two, the described algorithm was implemented for each route, considering the slope, and using the average speed for each study as the lower limit in the domain restrictions. Therefore, the aim was to verify that the system decreases or equalizes its energy consumption while developing a similar or higher speed by employing the proposed driving strategy, in other words, its range was optimized while the travel time was enhanced or remained unaffected.

3. Results

3.1. Components Characterization Results

The experimental data sets were subject to an outlier removal process, in which non-normality was identified for all of them; therefore, the Mahalanobis distance was calculated and outliers were identified and removed using the cumulative distribution function.

3.1.1. Charger

Figure 4a presents the charger test results. It shows the efficiency of the charger as well as the power drawn from the power grid (AC) and the power delivered to the battery (DC). The test allows the identification of the charging process stages, where: CC stage runs from 0% to 93% SoC developing an efficiency above 80%, with a maximum value of 90% when the battery reached maximum voltage; while CV stage runs from 93% to 100% SoC, developing an efficiency decreasing from 90% to 55% as the current decreased. Through the experiment, the total efficiency calculated for the charger was 85.77%.

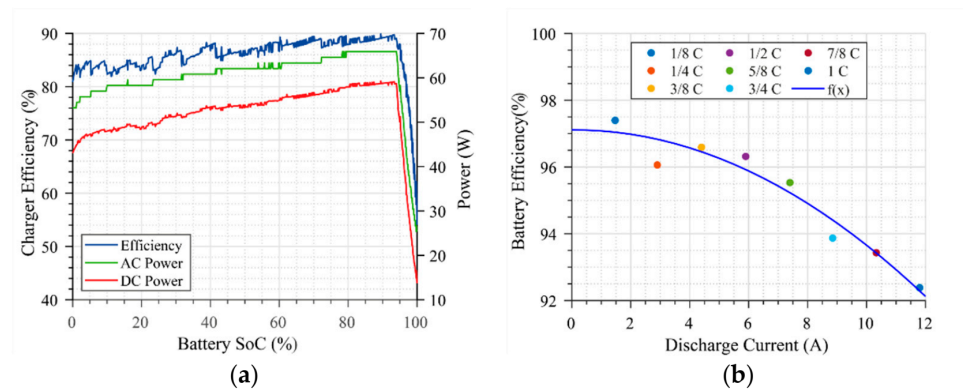


Figure 4. (a) Charger efficiency and power vs. battery state of charge (SoC). (b) Battery efficiency vs. discharge current.

3.1.2. Battery

Figure 4b presents the results of the battery efficiency experiment. It shows the battery efficiency for each C rate test. These results were fitted using a quadratic curve representing the battery efficiency function, which is expressed as:

$$\eta_{bat} = -0.035115I_{bat}^2 + 0.0006196I_{bat} + 97.11 \quad (40)$$

3.1.3. CMRGs Group

Figure 5a shows the CMRGs group efficiency map obtained experimentally. The

subsystem efficiency exceeds 65% at most of its operating points, and 80% was its maximum. Additionally, for each assist level, the data allowed for the identification of the angular speed trend in the output shaft of the reduction gear as a function of the torque, as shown in Figure 5b.

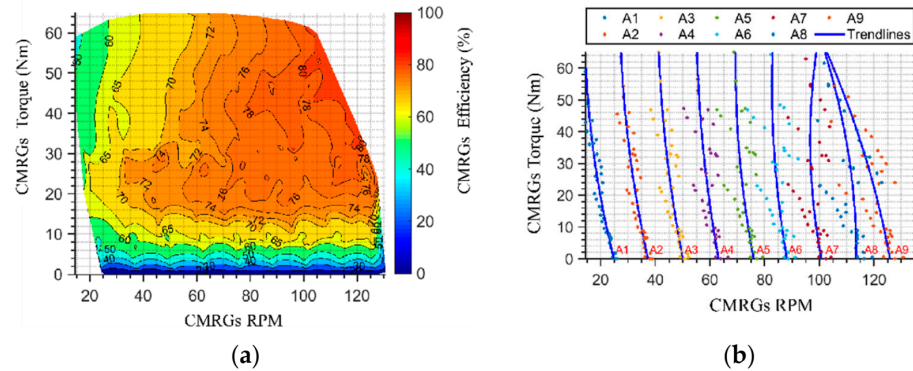


Figure 5. (a) CMRGs group efficiency map. (b) CMRGs group assist levels trendlines.

By implementing the linear regression method, the CMRGs efficiency map is represented for its torque and RPM using Equation (41), obtaining an R^2 of 0.944. Where C_m corresponds to constants for each term of the equation. (C_m values are shown in Appendix A). The parametric equations are graphically presented in Figure 6.

$$\eta_{CMRGs} = \sum_{m=1}^{20} C_m \times \sum_{i=0}^3 \sum_{j=0}^4 RPM_{crank_{spindle}}^i T_{CMRGs}^j \quad (41)$$

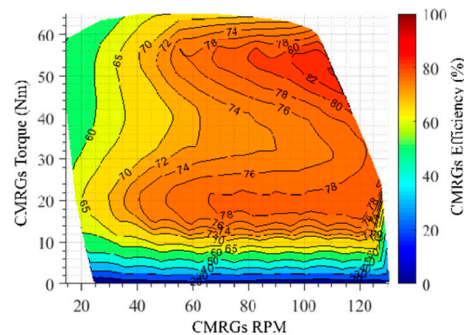


Figure 6. Efficiency map of the CMRGs represented by parametric equations.

3.1.4. E-Bike Efficiency Map

Below are global and partial (including all components except the charger) e-bike system efficiency maps. Although it is possible to generate as many efficiency maps as there are chain transmission ratios (TRs) in the system, efficiency maps are presented for the largest ratio, the smallest ratio, and an intermediate one. For each map, the chain transmission ratio is presented in the upper right-hand corner, where the first element refers to the teeth count for the chain wheel and the second one to the teeth count for the cassette. Figure 7a shows the e-bike's partial efficiency map for 34:21 transmission ratio. Figure 7b–d shows the global efficiency maps of the e-bike for different chain transmission ratios. The graphs are intersected with the curves representing the road load for different slope percentages; they show the increase in wheel torque as the system operates at a higher speed due to the drag force. The maximum efficiency for a global system map was 62% when delivering 480 W.

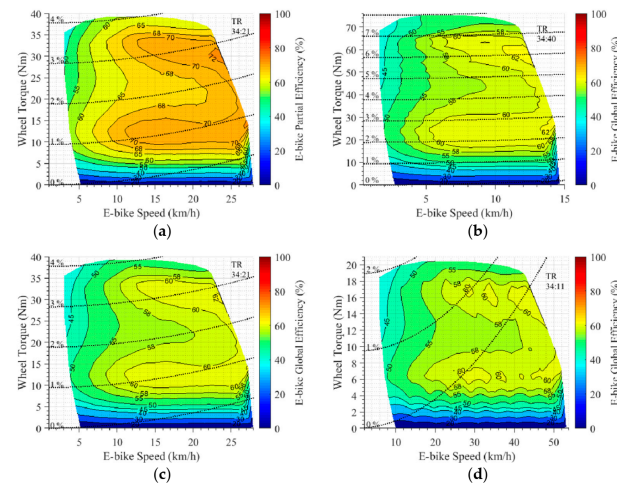


Figure 7. (a) E-bike partial efficiency map with 34:21 transmission ratio (TR). (b) E-bike global efficiency map with 34:40 TR. (c) E-bike global efficiency map with 34:21 TR. (d) E-bike global efficiency map with 34:11 TR.

3.1.5. Wh/km Consumption Map

The characterization of the components and their integration with the equations defining the movement of the system enable the generation of per unit of distance energy consumption maps for the system. These are shown in Figure 8 and are presented as a graphic tool that facilitates the understanding of the proposed driving strategy. The graphs are also intersected with the curves representing the road load for different slope percentages.

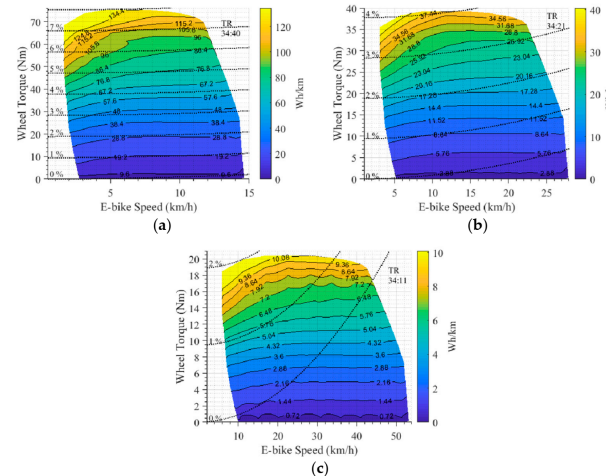


Figure 8. (a) Per unit of distance e-bike energy consumption map with 34:40 transmission ratio (TR). (b) Per unit of distance e-energy consumption map with 34:21 TR. (c) Per unit of distance e-bike energy consumption map with 34:11 TR.

3.2. Driving Strategies Results

Comparative results for the system performance under e-bike power alone for strategy two (cadence between 50 and 60) and strategy one (proposed in this article) are presented in Table 4. This shows that, by using the assistance level (AL) and chain transmission ratio indicated by strategy one for each route, energy consumption was reduced by up to 5.5% (1.8% slope). The system speed was also increased by up to 38.5% while electricity consumption was reduced (slope 8.8%).

Table 4. Performance of strategies two and one under e-bike power alone (E stands for energy, v stands for speed, TR_{chain} stands for chain transmission ratio and AL stands for assistance level).

Slope (%)	Cadence 50 < RPM < 60 AL 4			Configuration Used at Constant v		Wh/km Optimization Strategy			Algorithm Output	
	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL
0.0	72.0	0.0	24.1	34/11	4	72.0	0.0	24.1	34/11	4
1.8	112.3	0.0	15.6	34/17	4	106.1	0.0	15.6	34/13	3
3.5	172.8	0.0	9.9	34/27	4	164.6	0.0	10.6	34/19	3
5.2	243.4	0.0	9.7	34/27	4	236.6	0.0	10.3	34/19	3
7.0	314.9	0.0	8.4	34/31	4	308.2	0.0	9.2	34/21	3
8.8	386.9	0.0	6.5	34/40	4	380.6	0.0	9.0	34/21	3
10.5	458.9	0.0	6.4	34/40	4	450.7	0.0	7.8	34/24	3
12.3	534.2	0.0	7.2	34/35	4	516.0	0.0	7.7	34/24	3

Comparative results for the system's performance under e-bike and cyclist power for strategy two (cadence between 50 and 60) and strategy one (proposed in this article) are presented in Table 5. In this case, the system's performance under strategies one and two was the same. This means that the level of assistance and the chain transmission ratio indicated by strategy one for each route coincides with the behavior of strategy two. This also indicates that no operational point in the system was determined where electrical energy consumption would be reduced for the same or higher speeds.

Table 5. Performance of strategies two and one under e-bike and human power (E stands for energy, v stands for speed, TR_{chain} stands for chain transmission ratio and AL stands for assistance level).

Slope (%)	Cadence 50 < rpm < 60 AL 4			Configuration Used at Constant v		Wh/km Optimization Strategy			Algorithm Output	
	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL
0.0	24.5	39.7	25.2	34/11	4	24.5	39.7	25.2	34/11	4
1.8	82.6	41.6	24.0	34/11	4	82.6	41.6	24.0	34/11	4
3.5	104.6	56.5	17.7	34/15	4	104.6	56.5	17.7	34/15	4
5.2	130.1	78.5	12.7	34/21	4	130.1	78.5	12.7	34/21	4
7.0	179.5	90.6	11.0	34/24	4	179.5	90.6	11.0	34/24	4
8.8	231.4	102.8	9.7	34/27	4	231.4	102.8	9.7	34/27	4
10.5	302.4	104.5	9.6	34/27	4	302.4	104.5	9.6	34/27	4
12.3	300.0	151.1	6.6	34/40	4	300.0	151.1	6.6	34/40	4

Since the output of the two strategies was the same, new tests are conducted for the proposed driving strategy, modifying the speed limit in the algorithm developed. The speed restriction is set at 90% of the average speed resulting from the tests of strategy two (cadence between 50 and 60).

Comparative results for the system's performance under e-bike and cyclist power for strategy two (cadence between 50 and 60) and strategy one modifying the speed restriction (proposed in this article) are presented in Table 6. This shows that, by using the driving strategy proposed for each route, it was possible to reduce the energy consumption by up to 19% (slope 3.5%) while decreasing the speed by 11.3%. The results show that it is possible to reduce the electrical energy consumption of the system and thus improve its autonomy by giving up some of the system's speed.

Table 6. Performance of strategies two and one (speed restriction modified) under e-bike and human power (E stands for energy, v stands for speed, TR_{chain} stands for chain transmission ratio and AL stands for assistance level).

Slope (%)	Cadence 50 < rpm < 60 AL 4			Configuration Used at Constant v		Wh/km Optimization Strategy Using 90% of Avg v			Algorithm Output	
	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL
0.0	24.5	39.7	25.2	34/11	4	17.0	43.3	23.2	34/17	6
1.8	82.6	41.6	24.0	34/11	4	73.3	45.8	22.0	34/15	5
3.5	104.6	56.5	17.7	34/15	4	85.21	62.8	16.0	34/13	3
5.2	130.1	78.5	12.7	34/21	4	116.6	82.5	12.2	34/17	3
7.0	179.5	90.6	11.0	34/24	4	164.0	100.8	10.0	34/27	4
8.8	231.4	102.8	9.7	34/27	4	215.1	113.0	8.9	34/15	2
10.5	302.4	104.5	9.6	34/27	4	290.3	108.5	9.3	34/21	3
12.3	300.0	151.1	6.6	34/40	4	259.7	170.8	5.9	34/35	3

Comparative results for the system's performance under e-bike power alone for strategy three (cadence between 70 and 90) and strategy one (proposed in this article) are presented in Table 7. This shows that, by using the driving strategy proposed for each route, it was possible to reduce the energy consumption by up to 14.4% (slope 7.0%) while obtaining a higher speed. Likewise, it was possible to increase the speed of the system by up to 9.6% while reducing electrical energy consumption (slope 12.3%). For the cases of 0.0% and 1.8% slope routes, the system's performance was the same using both strategies.

Table 7. Performance of strategies three and one under e-bike power alone (E stands for energy, v stands for speed, TR_{chain} stands for chain transmission ratio and AL stands for assistance level).

Slope (%)	Cadence 70 < rpm < 90 AL 7			Configuration Used at Constant v		Wh/km Optimization Strategy			Algorithm Output	
	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL
0.0	202.5	0.0	38.1	34/11	7	202.5	0.0	38.1	34/11	7
1.8	295.6	0.0	37.7	34/11	7	295.6	0.0	37.7	34/11	7
3.5	275.5	0.0	30.0	34/11	7	273.6	0.0	30.0	34/11	6
5.2	321.6	0.0	20.2	34/15	7	290.4	0.0	20.4	34/15	5
7.0	386.4	0.0	15.7	34/27	7	330.7	0.0	16.3	34/15	4
8.8	486.2	0.0	17.5	34/31	7	464.6	0.0	18.9	34/19	6
10.5	552.4	0.0	13.6	34/31	7	492.9	0.0	14.4	34/21	5
12.3	650.8	0.0	13.6	34/31	7	610.0	0.0	14.9	34/24	6

Comparative results for the system's performance under e-bike and cyclist power for strategy three (cadence between 70 and 90) and strategy one (proposed in this article) are presented in Table 8. This shows that, by using the driving strategy proposed for each route, it was possible to reduce the energy consumption by up to 16.1% (slope 7.0%) while obtaining a higher speed. Likewise, it was possible to increase the speed of the system by up to 11.5% while reducing electrical energy consumption (slope 12.3%).

Table 8. Performance of strategies three and one under e-bike and human power (E stands for energy, v stands for speed, TR_{chain} stands for chain transmission ratio and AL stands for assistance level).

Slope (%)	Cadence 70 < rpm < 90 AL 7			Configuration Used at Constant v		Wh/km Optimization Strategy			Algorithm Output	
	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL	E_{elec} (Wh)	$E_{cyclist}$ (Wh)	Avg v (km/h)	TR_{chain}	AL
0.0	158.4	26.0	38.5	34/11	7	158.4	26.0	38.5	34/11	7
1.8	181.4	30.7	32.6	34/13	7	176.2	30.2	33.1	34/11	6
3.5	223.7	35.4	28.2	34/15	7	198.2	35.8	27.9	34/11	5
5.2	253.9	44.7	22.4	34/19	7	218.9	44.4	22.5	34/11	4
7.0	295.2	56.2	17.8	34/24	7	247.7	54.6	18.3	34/11	4
8.8	359.0	63.4	15.8	34/27	7	313.9	60.7	16.5	34/15	4
10.5	445.0	63.9	15.6	34/27	7	406.6	57.5	17.4	34/17	5
12.3	505.9	73.4	13.6	34/31	7	447.4	68.3	14.6	34/21	5

4. Discussion

This article presented a driving strategy focused on enhancing the range of mid-drive motor electric bicycles, based on the proper selection of the chain transmission ratio and the assistance level. For this purpose, the system's dynamic equations were used together with the characterization of the bicycle components' efficiency performance, thus, determining per unit of distance energy consumption for different operational combinations of the system, and thereby selecting the operational point that allows developing the longest range.

The global efficiency map of a mid-drive motor e-bike was generated through the integration of the efficiency of its subsystems. This showed that the system does not have a punctual efficiency, but that this aspect varies continuously throughout all its operating range. The components characterization results demonstrated that every subsystem has a significant impact on the overall efficiency, this aspect being highly determined by the CMRGs sub-system, which has the greatest variation (between 20% and 80%). In addition, the system efficiency is highly affected by the chain drive, with efficiency varying between 70% and 98% depending on chain tension. Components such as the battery (efficiency varying between 88% and 97% according to the discharge current) and the charger (which has 85.7% total efficiency) have fewer impacts on the system's efficiency. The characterization of the per unit of distance energy consumption facilitates the system's range enhancement. By contrasting the per unit of distance energy consumption maps, it is possible to determine that the range of the system is extended by using the chain transmission ratios that increase speed. Likewise, these maps show that, for the same slope, the greater the speed the greater the energy consumption due to the drag force increase.

It was found that in order to increase the system's range, focusing on its per unit of distance energy consumption is more relevant than focusing on its operational efficiency. This happens because, whilst the operational efficiency evaluates the system's internal components performance alone, the per unit of distance analysis considers both the internal components' performance as well as the system interaction with external forces, making it possible to completely identify the use given to the energy to be transformed into displacement.

The efficiency differences and per unit of distance energy consumption analyses are contrasted in the maps that represent them (Figures 7 and 8). Where the best efficiency operational points are generated at high speeds and medium to high torques, while the best energy consumption points per unit of distance are generated at low torques, regardless of speed.

Driving strategies performance were characterized for their energy consumption and average speed, observing that, for the driving strategies comparative results, the driving strategy proposed in this article equals or increases the range of the system in contrast to the driving strategies commonly used by untrained cyclists [37,38].

This article is presented as a new advance in the field of electric bicycle range optimization by carrying out a complete analysis of the variables affecting an e-bike performance

and electrical consumption. Likewise, the algorithm proposed is presented as a tool that offers suggestions to the cyclist to enhance the system's range from the selection of the chain transmission ratio and the level of assistance according to the slope. The developed research differs from the previous methodology (developed by De la Iglesia et. al. [17] and used to increase the range of electric bicycles through neural networks), by implementing the mathematical model of the system and the performance of the efficiency of its components to generate the optimal selection of the assist level and the chain transmission ratio.

Improving the electric bicycle range is a complex matter, since it encompasses multiple variables, such as the desired travel time, the route profile, the power delivered by the cyclist, the wind speed, among others, making the best way to ride a bicycle not a single solution. Future work is expected to implement road tests and incorporate more variables to the problem, covering objectives such as traveling a variable slopes route in a given time by minimizing energy consumption or determining the system operation to complete a route by using the remaining capacity of the battery.

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Appendix A

Table A1. Terms of the controller, motor, and reduction gears (CMRGs) efficiency parametric equation.

Parametric Equation Coefficients					
C1	−5.2434	C8	−0.0026924	C15	0
C2	6.8388	C9	5.4134×10^{-5}	C16	4.6057×10^{-5}
C3	−0.30491	C10	-3.1852×10^{-7}	C17	0
C4	0.005517	C11	−0.011524	C18	0
C5	-3.6145×10^{-5}	C12	−0.00017694	C19	0
C6	0.86255	C13	9.4452×10^{-6}	C20	0
C7	0.045169	C14	-1.2101×10^{-7}		

References

1. Kheirandish, A.; Kazemi, M.S.; Dahari, M. Dynamic Performance Assessment of the Efficiency of Fuel Cell-Powered Bicycle: An Experimental Approach. *Int. J. Hydrogen Energy* **2014**, *39*, 13276–13284. [\[CrossRef\]](#)
2. Zhang, S.; Wu, Y.; Un, P.; Fu, L.; Hao, J. Modeling Real-World Fuel Consumption and Carbon Dioxide Emissions with High Resolution for Light-Duty Passenger Vehicles in a Traffic Populated City. *Energy* **2016**, *113*, 461–471. [\[CrossRef\]](#)
3. Hawkins, T.R.; Singh, B.; Majeau-Bettez, G.; Strømman, A.H. Comparative Environmental Life Cycle Assessment of Conventional and Electric Vehicles. *J. Ind. Ecol.* **2013**, *17*, 53–64. [\[CrossRef\]](#)
4. Salmeron-Manzano, E.; Manzano-Agugliaro, F. The Electric Bicycle: Worldwide Research Trends. *Energies* **2018**, *11*, 1894. [\[CrossRef\]](#)
5. Ferreira, J.C.; Monteiro, V.; Afonso, J.A.; Afonso, J.L. Mobile Cockpit System for Enhanced Electric Bicycle Use. *IEEE Trans. Ind. Inform.* **2015**, *11*, 1017–1027. [\[CrossRef\]](#)
6. Popovich, N.; Gordon, E.; Shao, Z.; Xing, Y.; Wang, Y.; Handy, S. Experiences of Electric Bicycle Users in the Sacramento, California Area. *Travel Behav. Soc.* **2014**, *1*, 37–44. [\[CrossRef\]](#)
7. Liu, K.; Wang, J.; Yamamoto, T.; Morikawa, T. Modelling the Multilevel Structure and Mixed Effects of the Factors Influencing the Energy Consumption of Electric Vehicles. *Appl. Energy* **2016**, *183*, 1351–1360. [\[CrossRef\]](#)

8. Husain, I. *Electric and Hybrid Vehicles: Design Fundamentals*; CRC Press: Boca Raton, FL, USA, 2011.
9. Muetze, A.; Tan, Y. Electric Bicycles—A Performance Evaluation. *IEEE Ind. Appl. Mag.* **2007**, *13*, 12–21. [\[CrossRef\]](#)
10. Evtimov, I.; Ivanov, R.; Staneva, G.; Kadikyanov, G. A Study on Electric Bicycle Energy Efficiency. *Transp. Probl.* **2017**, *10*, 131–140. [\[CrossRef\]](#)
11. Kang, D.K.; Kim, M.S. Hardware-in-the-Loop Simulation to Evaluate the Drive Performance of the Electric Two-Wheelers on a Motor Dynamometer. *Int. J. Automot. Technol.* **2015**, *16*, 1031–1040. [\[CrossRef\]](#)
12. Hung, N.B.; Jaewon, S.; Lim, O. A Study of the Effects of Input Parameters on the Dynamics and Required Power of an Electric Bicycle. *Appl. Energy* **2017**, *204*, 1347–1362. [\[CrossRef\]](#)
13. Hung, N.B.; Sung, J.; Lim, O. A Simulation and Experimental Study of Operating Performance of an Electric Bicycle Integrated with a Semi-Automatic Transmission. *Appl. Energy* **2018**, *221*, 319–333. [\[CrossRef\]](#)
14. Hung, N.B.; Lim, O. The Effects of Operating Conditions and Structural Parameters on the Dynamic, Electric Consumption and Power Generation Characteristics of an Electric Assisted Bicycle. *Appl. Energy* **2019**, *247*, 285–296. [\[CrossRef\]](#)
15. Arango, I.; Godoy, A.; Lopez, C. E-Bikes for Steep Roads: Mid Drive and Hub Drive Motor Efficiency Comparison. *Int. J. Veh. Syst. Model. Test.* **2018**, *13*, 44–58. [\[CrossRef\]](#)
16. Gebhard, L.; Golab, L.; Keshav, S.; Demeer, H. Range Prediction for Electric Bicycles. In Proceedings of the eventh International Conference on Future Energy Systems e-Energy, Waterloo, ON, Canada, 21–24 June 2016; pp. 224–234. [\[CrossRef\]](#)
17. De La Iglesia, D.H.; Villarubia, G.; De Paz, J.F.; Bajo, J. Multi-Sensor Information Fusion for Optimizing Electric Bicycle Routes Using a Swarm Intelligence Algorithm. *Sensors* **2017**, *17*, 2501. [\[CrossRef\]](#)
18. Minav, T.A.; Heikkinen, J.E.; Pietola, M. Direct Driven Hydraulic Drive for New Powertrain Topologies for Non-Road Mobile Machinery. *Electr. Power Syst. Res.* **2017**, *152*, 390–400. [\[CrossRef\]](#)
19. Wu, J.; Liang, J.; Ruan, J.; Zhang, N.; Walker, P.D. Efficiency Comparison of Electric Vehicles Powertrains with Dual Motor and Single Motor Input. *Mech. Mach. Theory* **2018**, *128*, 569–585. [\[CrossRef\]](#)
20. Sun, B.; Gao, S.; Ma, C. Mathematical Methods Applied to Economy Optimization of an Electric Vehicle with Distributed Power Train System. *Math. Probl. Eng.* **2016**, *2016*, 4949561. [\[CrossRef\]](#)
21. Wang, H.; Dusmez, S.; Khaligh, A. Design and Analysis of a Full-Bridge LLC-Based PEV Charger Optimized for Wide Battery Voltage Range. *IEEE Trans. Veh. Technol.* **2014**, *63*, 1603–1613. [\[CrossRef\]](#)
22. Vu, H.-N.; Choi, W. A Novel Dual Full-Bridge LLC Resonant Converter for CC and CV Charges of Batteries for Electric Vehicles. *IEEE Trans. Ind. Electron.* **2018**, *65*, 2212–2225. [\[CrossRef\]](#)
23. Copetti, J.; Lorenzo, E.; Chenlo, F. A General Battery Model for PV System Simulation. *Prog. Photovolt. Res. Appl.* **1993**, *1*, 283–292. [\[CrossRef\]](#)
24. Liu, Y.-C.; Chang, S.-B. Design and Implementation of a Smart Lithium-Ion Battery Capacity Estimation System for E-Bike. *World Electr. Veh. J.* **2010**, *4*, 370–378. [\[CrossRef\]](#)
25. Hung, N.B.; Lim, O. A Review of History, Development, Design and Research of Electric Bicycles. *Appl. Energy* **2020**, *260*, 114323. [\[CrossRef\]](#)
26. Farhad, S.; Nazari, A. Introducing the Energy Efficiency Map of Lithium-Ion Batteries. *Int. J. Energy Res.* **2019**, *43*, 931–944. [\[CrossRef\]](#)
27. Scarabelot, L.T.; Rambo, C.R.; Rampinelli, G.A. A Relative Power-Based Adaptive Hybrid Model for DC/AC Average Inverter Efficiency of Photovoltaics Systems. *Renew. Sustain. Energy Rev.* **2018**, *92*, 470–477. [\[CrossRef\]](#)
28. Ding, X.; Du, M.; Zhou, T.; Guo, H.; Zhang, C. Comprehensive Comparison between Silicon Carbide MOSFETs and Silicon IGBTs Based Traction Systems for Electric Vehicles. *Appl. Energy* **2017**, *194*, 626–634. [\[CrossRef\]](#)
29. Hashemnia, N.; Asaei, B. Comparative Study of Using Different Electric Motors in the Electric Vehicles. In Proceedings of the 2008 18th International Conference on Electrical Machines, Vilamoura, Portugal, 6–9 September 2008; pp. 1–5. [\[CrossRef\]](#)
30. SPINNINGMAGNETS. Electricbike. Available online: <https://www.electricbike.com/bafang-ultra-max/> (accessed on 4 August 2020).
31. GrinTechnologies. ebikes.ca. Available online: <https://www.ebikes.ca/learn/hub-motors.html> (accessed on 4 August 2020).
32. Li, X.; Sosa, M.; Andersson, M.; Olofsson, U. A Study of the Efficiency of Spur Gears Made of Powder Metallurgy Materials—Ground versus Super-Finished Surfaces. *Tribol. Int.* **2016**, *95*, 211–220. [\[CrossRef\]](#)
33. Baglioni, S.; Cianetti, F.; Landi, L. Influence of the Addendum Modification on Spur Gear Efficiency. *Mech. Mach. Theory* **2012**, *49*, 216–233. [\[CrossRef\]](#)
34. Spicer, J.B. Effects of the Nonlinear Elastic Behavior of Bicycle Chain on Transmission Efficiency. *J. Appl. Mech.* **2013**, *80*. [\[CrossRef\]](#)
35. Spicer, J.B.; Richardson, C.J.K.; Ehrlich, M.J.; Bernstein, J.R.; Fukuda, M.; Terada, M. Effects of Frictional Loss on Bicycle Chain Drive Efficiency. *J. Mech. Des.* **2001**, *123*, 598–605. [\[CrossRef\]](#)
36. Hollingworth, N.E.; Hills, D.A. Theoretical Efficiency of a Cranked Link Chain Drive. *Proc. Inst. Mech. Eng. Part. C J. Mech. Eng. Sci.* **1986**, *200*, 375–377. [\[CrossRef\]](#)
37. Marsh, A.P.; Martin, P.E.; Foley, K.O. Effect of Cadence, Cycling Experience, and Aerobic Power on Delta Efficiency during Cycling. *Delta* **2000**, *32*, 1630–1634. [\[CrossRef\]](#) [\[PubMed\]](#)
38. Grossoleil, D.; Meizel, D. *Modelling the Hybridisation of Human and Artificial Energy Applied at an Electrical Bicycle*; IFAC: Geneva, Switzerland, 2010; Volume 11. [\[CrossRef\]](#)